

3.4 Equations for thermoelasticity for isotropic materials

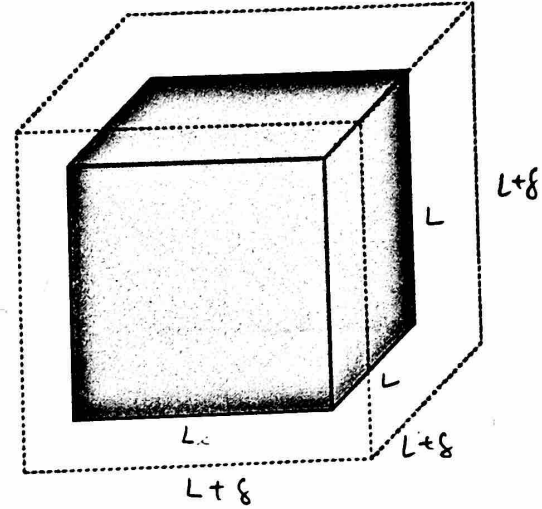
①

$$\text{Heat } T_1 \rightarrow T_2 \\ \Delta T = T_2 - T_1$$

For body shown, if ΔT was applied

\Rightarrow Expansion by δ_T " is equal in all directions "

$$\delta_T = L \alpha \Delta T \text{ where } \alpha: \text{coefficient of thermal expansion}$$



$$\text{thermal strain } \epsilon_T = \frac{\delta_T}{L} = \alpha \Delta T$$

* For mechanical and thermo-mechanical loading of isotropic materials

$$\Rightarrow \text{Total Strain} = \epsilon_{\text{mechanical}} + \epsilon_{\text{thermal}}$$

Thus

$$\epsilon_{xx} = \frac{1}{E} (\sigma_{xx} - \nu(\sigma_{yy} + \sigma_{zz})) + \alpha \Delta T$$

$$\epsilon_{yy} = \frac{1}{E} (\sigma_{yy} - \nu(\sigma_{xx} + \sigma_{zz})) + \alpha \Delta T$$

$$\epsilon_{zz} = \frac{1}{E} (\sigma_{zz} - \nu(\sigma_{xx} + \sigma_{yy})) + \alpha \Delta T$$

$$\epsilon_{xy} = \frac{\sigma_{xy}}{2G}, \quad \epsilon_{yz} = \frac{\sigma_{yz}}{2G}, \quad \epsilon_{xz} = \frac{\sigma_{xz}}{2G}$$

$$\Rightarrow \{\epsilon\} = [S] \{\sigma\} \Rightarrow \{\sigma\} = [S]^{-1} \{\epsilon\}$$

$$\sigma_{xx} = \lambda e + 2G \epsilon_{xx} - C \Delta T$$

$$e = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}$$

$$\sigma_{yy} = \lambda e + 2G \epsilon_{yy} - C \Delta T$$

$$C = (3\lambda + 2G)\alpha = \frac{E\alpha}{1 - 2\nu}$$

$$\sigma_{zz} = \lambda e + 2G \epsilon_{zz} - C \Delta T$$

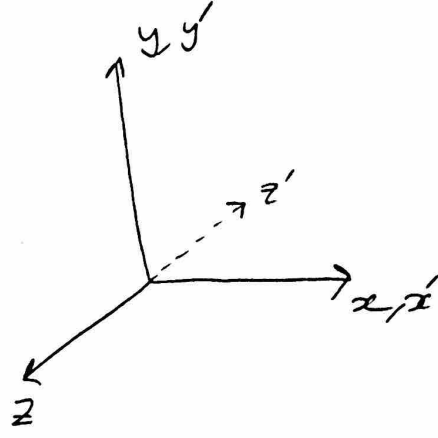
$$\sigma_{xy} = 2G \epsilon_{xy}, \quad \sigma_{xz} = 2G \epsilon_{xz}, \quad \sigma_{yz} = 2G \epsilon_{yz}$$

3.5 Hooke's Law: Orthotropic Materials

- Orthotropic materials have 3 orthogonal planes of material symmetry and 3 corresponding orthogonal axes.

- Examples: Wood, reinforced concrete and laminated composites.

* let (x, y, z) original coordinate system
and (x', y', z') new = = .
 $(x, y) \rightarrow$ plane of symmetry



Transformation matrix

$$[Q] = \begin{bmatrix} l_1 & m_1 & n_1 \\ l_2 & m_2 & n_2 \\ l_3 & m_3 & n_3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & -1 \end{bmatrix}$$

If $[\sigma] = \begin{bmatrix} \sigma_{xx} & \sigma_{xy} & \sigma_{xz} \\ \sigma_{xy} & \sigma_{yy} & \sigma_{yz} \\ \sigma_{xz} & \sigma_{yz} & \sigma_{zz} \end{bmatrix}$, we can find $[\sigma'] = [Q][\sigma][Q]^T$

$$\Rightarrow \sigma'_{xx} = \sigma_{xx} , \sigma'_{yy} = \sigma_{yy} , \sigma'_{zz} = \sigma_{zz} , \sigma'_{xy} = \sigma_{xy}$$

$$\sigma'_{xz} = -\sigma_{xz} , \sigma'_{yz} = -\sigma_{yz} \quad \text{--- Eq (1)}$$

and

$$\epsilon'_{xx} = \epsilon_{xx} , \epsilon'_{yy} = \epsilon_{yy} , \epsilon'_{zz} = \epsilon_{zz} , \epsilon'_{xy} = \epsilon_{xy}$$

$$\epsilon'_{xz} = -\epsilon_{xz} , \epsilon'_{yz} = -\epsilon_{yz} \quad \text{--- Eq (2)}$$

Remember, Hooke's Law
for Anisotropic material

$$\sigma_{xx} = C_{11} \epsilon_{xx} + C_{12} \epsilon_{yy} + C_{13} \epsilon_{zz} + 2C_{14} \epsilon_{xy} + 2C_{15} \epsilon_{xz} + 2C_{16} \epsilon_{yz}$$

$$\sigma_{yy} = C_{21} \epsilon_{xx} + C_{22} \epsilon_{yy} + C_{23} \epsilon_{zz} + 2C_{24} \epsilon_{xy} + 2C_{25} \epsilon_{xz} + 2C_{26} \epsilon_{yz}$$

$$\sigma_{zz} = C_{31} \epsilon_{xx} + C_{32} \epsilon_{yy} + C_{33} \epsilon_{zz} + 2C_{34} \epsilon_{xy} + 2C_{35} \epsilon_{xz} + 2C_{36} \epsilon_{yz}$$

$$\sigma_{xy} = C_{41} \epsilon_{xx} + C_{42} \epsilon_{yy} + C_{43} \epsilon_{zz} + 2C_{44} \epsilon_{xy} + 2C_{45} \epsilon_{xz} + 2C_{46} \epsilon_{yz}$$

$$\sigma_{xz} = C_{51} \epsilon_{xx} + C_{52} \epsilon_{yy} + C_{53} \epsilon_{zz} + 2C_{54} \epsilon_{xy} + 2C_{55} \epsilon_{xz} + 2C_{56} \epsilon_{yz}$$

$$\sigma_{yz} = C_{61} \epsilon_{xx} + C_{62} \epsilon_{yy} + C_{63} \epsilon_{zz} + 2C_{64} \epsilon_{xy} + 2C_{65} \epsilon_{xz} + 2C_{66} \epsilon_{yz}$$

For transformed stress state

$$\sigma'_{xx} = C_{11} \epsilon'_{xx} + C_{12} \epsilon'_{yy} + C_{13} \epsilon'_{zz} + 2C_{14} \epsilon'_{xy} + 2C_{15} \epsilon'_{xz} + 2C_{16} \epsilon'_{yz}$$

But From Eq(1), and Eq(2)

$$\sigma_{xx} = \sigma'_{xx} = C_{11} \epsilon_{xx} + C_{12} \epsilon_{yy} + C_{13} \epsilon_{zz} + 2C_{14} \epsilon_{xy} - 2C_{15} \epsilon_{yz} - 2C_{16} \epsilon_{yz}$$

That means $\Rightarrow C_{15} = -C_{15} = 0$ and $C_{16} = -C_{16} = 0$

Do this for all $\sigma_{ij} \Rightarrow C_{25} = -C_{25} = 0, C_{26} = -C_{26} = 0$

$C_{35} = -C_{35} = 0, C_{36} = -C_{36} = 0, C_{45} = -C_{45} = 0, C_{46} = -C_{46} = 0$

\Rightarrow Hooke's Law (matrix form)

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{yz} \\ \sigma_{yz} \end{Bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & C_{14} & 0 & 0 \\ C_{12} & C_{22} & C_{23} & C_{24} & 0 & 0 \\ C_{13} & C_{23} & C_{33} & C_{34} & 0 & 0 \\ C_{14} & C_{24} & C_{34} & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ 2\epsilon_{xy} \\ 2\epsilon_{xz} \\ 2\epsilon_{yz} \end{Bmatrix}$$

⇒ For a material with one plane of symmetry
 ⇒ we require twelve (12) elastic constants.

* For orthotropic material, we have two additional planes of symmetry (x,z) and (y,z) ⇒ total 3 planes of symmetry.

* In this case, the transformation matrix [Q], becomes

$$[Q] = \begin{bmatrix} l_1 & m_1 & n_1 \\ l_2 & m_2 & n_2 \\ l_3 & m_3 & n_3 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

* Following the same procedure, Hooke's Law becomes

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{xy} \\ \sigma_{xz} \\ \sigma_{yz} \end{Bmatrix} = \begin{bmatrix} C_{11} & C_{12} & C_{13} & 0 & 0 & 0 \\ C_{21} & C_{22} & C_{23} & 0 & 0 & 0 \\ C_{31} & C_{32} & C_{33} & 0 & 0 & 0 \\ 0 & 0 & 0 & C_{44} & 0 & 0 \\ 0 & 0 & 0 & 0 & C_{55} & 0 \\ 0 & 0 & 0 & 0 & 0 & C_{66} \end{bmatrix} \begin{Bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{xy} \\ \epsilon_{xz} \\ \epsilon_{yz} \end{Bmatrix}$$

For an orthotropic material (material with 3 planes of symmetry), we need 9 (nine) elastic coefficients.

- E_x, E_y, E_z ⇒ Elastic moduli in x, y and z direction.
- G_{xy}, G_{xz}, G_{yz} ⇒ Shear moduli in (x,y), (x,z) and (y,z) planes.
- $\nu_{xy}, \nu_{xz}, \nu_{yz}$ ⇒ Poisson's ratios in (x,y), (x,z) and (y,z) planes.

Hooke's Law, $\sigma = f(\epsilon)$

(5)

$$\begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{yz} \\ \sigma_{zx} \\ \sigma_{xy} \end{bmatrix} = \begin{bmatrix} \frac{1-\nu_{yz}\nu_{zy}}{E_y E_z \Delta} & \frac{\nu_{yx} + \nu_{zx}\nu_{yz}}{E_y E_z \Delta} & \frac{\nu_{zx} + \nu_{yx}\nu_{zy}}{E_y E_z \Delta} & 0 & 0 & 0 \\ \frac{\nu_{xy} + \nu_{xz}\nu_{zy}}{E_z E_x \Delta} & \frac{1-\nu_{zx}\nu_{xz}}{E_z E_x \Delta} & \frac{\nu_{zy} + \nu_{zx}\nu_{xy}}{E_z E_x \Delta} & 0 & 0 & 0 \\ \frac{\nu_{xz} + \nu_{xy}\nu_{yz}}{E_x E_y \Delta} & \frac{\nu_{yz} + \nu_{xz}\nu_{yx}}{E_x E_y \Delta} & \frac{1-\nu_{xy}\nu_{yx}}{E_x E_y \Delta} & 0 & 0 & 0 \\ 0 & 0 & 0 & 2G_{yz} & 0 & 0 \\ 0 & 0 & 0 & 0 & 2G_{zx} & 0 \\ 0 & 0 & 0 & 0 & 0 & 2G_{xy} \end{bmatrix} \begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{yz} \\ \epsilon_{zx} \\ \epsilon_{xy} \end{bmatrix}$$

$$\Delta = \frac{1 - \nu_{xy}\nu_{yx} - \nu_{yz}\nu_{zy} - \nu_{zx}\nu_{xz} - 2\nu_{xy}\nu_{yz}\nu_{zx}}{E_x E_y E_z}$$

$$\begin{cases} \frac{\nu_{yx} + \nu_{zx}\nu_{yz}}{E_y E_z \Delta} = \frac{\nu_{xy} + \nu_{xz}\nu_{zy}}{E_z E_x \Delta} \\ \frac{\nu_{zy} + \nu_{zx}\nu_{xy}}{E_z E_x \Delta} = \frac{\nu_{yz} + \nu_{xz}\nu_{yx}}{E_x E_y \Delta} \\ \frac{\nu_{zx} + \nu_{yx}\nu_{zy}}{E_y E_z \Delta} = \frac{\nu_{xz} + \nu_{xy}\nu_{yz}}{E_x E_y \Delta} \end{cases}$$

} To satisfy symmetry

Hooke's Law $\epsilon = f(\sigma)$

$$\begin{bmatrix} \epsilon_{xx} \\ \epsilon_{yy} \\ \epsilon_{zz} \\ \epsilon_{yz} \\ \epsilon_{zx} \\ \epsilon_{xy} \end{bmatrix} = \begin{bmatrix} \frac{1}{E_x} & -\frac{\nu_{yx}}{E_y} & -\frac{\nu_{zx}}{E_z} & 0 & 0 & 0 \\ -\frac{\nu_{xy}}{E_x} & \frac{1}{E_y} & -\frac{\nu_{zy}}{E_z} & 0 & 0 & 0 \\ -\frac{\nu_{xz}}{E_x} & -\frac{\nu_{yz}}{E_y} & \frac{1}{E_z} & 0 & 0 & 0 \\ 0 & 0 & 0 & \frac{1}{2G_{yz}} & 0 & 0 \\ 0 & 0 & 0 & 0 & \frac{1}{2G_{zx}} & 0 \\ 0 & 0 & 0 & 0 & 0 & \frac{1}{2G_{xy}} \end{bmatrix} \begin{bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \sigma_{zz} \\ \sigma_{yz} \\ \sigma_{zx} \\ \sigma_{xy} \end{bmatrix}$$

$$\frac{\nu_{yz}}{E_y} = \frac{\nu_{zy}}{E_z}, \quad \frac{\nu_{zx}}{E_z} = \frac{\nu_{xz}}{E_x}, \quad \frac{\nu_{xy}}{E_x} = \frac{\nu_{yx}}{E_y}$$

} For symmetry

(6)

For plane stress ($\sigma_{zz} = \sigma_{xz} = \sigma_{yz} = 0$)

$$\sigma_{xx} = \frac{E_x}{1 - \nu_{xy}\nu_{yx}} (\epsilon_{xx} + \nu_{yx}\epsilon_{yy})$$

$$\sigma_{yy} = \frac{E_y}{1 - \nu_{xy}\nu_{yx}} (\epsilon_{yy} + \nu_{xy}\epsilon_{xx})$$

$$\sigma_{xy} = 2G\epsilon_{xy}$$

strains

$$\epsilon_{xx} = \frac{\sigma_{xx}}{E_x} - \frac{\nu_{yx}\sigma_{yy}}{E_y}$$

$$\epsilon_{yy} = \frac{\sigma_{yy}}{E_y} - \nu_{xy} \frac{\sigma_{xx}}{E_x}$$

$$\epsilon_{zz} = - \left(\nu_{xz} \frac{\sigma_{xx}}{E_x} + \nu_{yz} \frac{\sigma_{yy}}{E_y} \right)$$

$$\epsilon_{xz} = \frac{1}{2G} \sigma_{xy}, \quad \epsilon_{yz} = \epsilon_{xy} = 0$$

Plane strain ? Practice Problem.