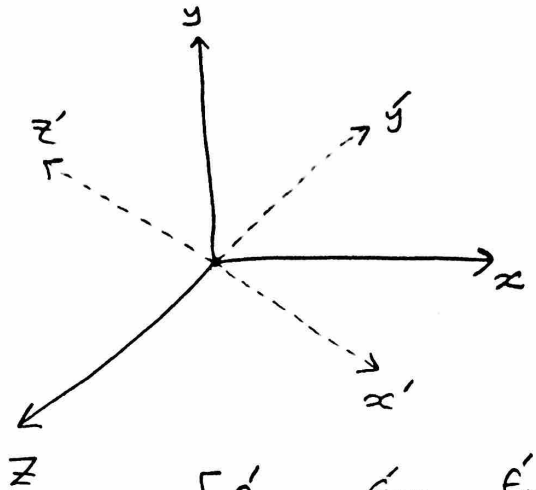


2.7.4 Strain transformations

①

$$[E] = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} \end{bmatrix}$$



To find the transformed strain state $[E']$

$$[E'] = [Q][E][Q^T],$$

$$[E'] = \begin{bmatrix} \epsilon'_{xx} & \epsilon'_{xy} & \epsilon'_{xz} \\ \epsilon'_{xy} & \epsilon'_{yy} & \epsilon'_{yz} \\ \epsilon'_{xz} & \epsilon'_{yz} & \epsilon'_{zz} \end{bmatrix}$$

$[Q]$ is the transformation matrix

$$[Q] = \begin{bmatrix} \cos(x', x) & \cos(x', y) & \cos(x', z) \\ \cos(y', x) & \cos(y', y) & \cos(y', z) \\ \cos(z', x) & \cos(z', y) & \cos(z', z) \end{bmatrix}, \quad [Q] = \begin{bmatrix} l_1 & m_1 & n_1 \\ l_2 & m_2 & n_2 \\ l_3 & m_3 & n_3 \end{bmatrix}$$

* Some properties of $[Q]$

① $[Q]^{-1} = [Q]^T \Rightarrow [Q]^{-1}[Q] = [Q][Q]^T = [I]$

② $\left. \begin{aligned} l_1^2 + l_2^2 + l_3^2 &= 1 \\ m_1^2 + m_2^2 + m_3^2 &= 1 \\ n_1^2 + n_2^2 + n_3^2 &= 1 \end{aligned} \right\} \begin{aligned} l_1 l_2 + m_1 m_2 + n_1 n_2 &= 0 \\ l_1 l_3 + m_1 m_3 + n_1 n_3 &= 0 \\ l_2 l_3 + m_2 m_3 + n_2 n_3 &= 0 \end{aligned} \left\{ \begin{aligned} l_i^2 + m_i^2 + n_i^2 &= 1 \\ l_2^2 + m_2^2 + n_2^2 &= 1 \\ l_3^2 + m_3^2 + n_3^2 &= 1 \end{aligned} \right. \Rightarrow l_i^2 + m_i^2 + n_i^2 = 1 \quad (i=1,2,3)$

③ $l_1 m_1 + l_2 m_2 + l_3 m_3 = 0 \Rightarrow \sum_{i=1}^3 l_i m_i = 0$

$l_1 n_1 + l_2 n_2 + l_3 n_3 = 0 \Rightarrow \sum_{i=1}^3 l_i n_i = 0$

$m_1 n_1 + m_2 n_2 + m_3 n_3 = 0 \Rightarrow \sum_{i=1}^3 m_i n_i = 0$

From $[\epsilon'] = [Q][\epsilon][Q]^T$ ②

$$\epsilon'_{xx} = l_1^2 \epsilon_{xx} + m_1^2 \epsilon_{yy} + n_1^2 \epsilon_{zz} + 2l_1 m_1 \epsilon_{xy} + 2l_1 n_1 \epsilon_{xz} + 2m_1 n_1 \epsilon_{yz}$$

$$\epsilon'_{yy} = l_2^2 \epsilon_{xx} + m_2^2 \epsilon_{yy} + n_2^2 \epsilon_{zz} + 2l_2 m_2 \epsilon_{xy} + 2l_2 n_2 \epsilon_{xz} + 2m_2 n_2 \epsilon_{yz}$$

$$\epsilon'_{zz} = l_3^2 \epsilon_{xx} + m_3^2 \epsilon_{yy} + n_3^2 \epsilon_{zz} + 2l_3 m_3 \epsilon_{xy} + 2l_3 n_3 \epsilon_{xz} + 2m_3 n_3 \epsilon_{yz}$$

$$\begin{aligned} \epsilon'_{xy} = \frac{1}{2} \gamma'_{xy} &= l_1 l_2 \epsilon_{xx} + m_1 m_2 \epsilon_{yy} + n_1 n_2 \epsilon_{zz} + (l_1 m_2 + l_2 m_1) \epsilon_{xy} \\ &+ (m_1 n_2 + m_2 n_1) \epsilon_{yz} + (l_1 n_2 + l_2 n_1) \epsilon_{xz} \end{aligned}$$

$$\begin{aligned} \epsilon'_{yz} = \frac{1}{2} \gamma'_{yz} &= l_2 l_3 \epsilon_{xx} + m_2 m_3 \epsilon_{yy} + n_2 n_3 \epsilon_{zz} + (l_2 m_3 + l_3 m_2) \epsilon_{xy} \\ &+ (m_2 n_3 + m_3 n_2) \epsilon_{yz} + (l_2 n_3 + l_3 n_2) \epsilon_{xz} \end{aligned}$$

$$\begin{aligned} \epsilon'_{xz} = \frac{1}{2} \gamma'_{xz} &= l_1 l_3 \epsilon_{xx} + m_1 m_3 \epsilon_{yy} + n_1 n_3 \epsilon_{zz} + (l_1 m_3 + l_3 m_1) \epsilon_{xy} \\ &+ (m_1 n_3 + m_3 n_1) \epsilon_{yz} + (l_1 n_3 + l_3 n_1) \epsilon_{xz} \end{aligned}$$

Just like stresses.

27.4 * Principal strains and principal directions

(3)

For a strain state $[\epsilon]$, we can find the principal values ϵ_i ($i=1,2,3$) and associated principal directions $\{s_i\}$ ($i=1,2,3$) as:-

$$[\epsilon]\{s_i\} = \epsilon_i\{s_i\}, \quad [\epsilon] = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} \end{bmatrix}$$

To find ϵ_i and $\{s_i\}$

$$[\epsilon]\{s_i\} - \epsilon_i\{s_i\} = 0 \Rightarrow \underbrace{([\epsilon] - \epsilon_i[I])}_{\det()=0} \underbrace{\{s_i\}}_{\neq 0} = 0$$

$$([\epsilon] - \epsilon_i[I]) = \begin{bmatrix} \epsilon_{xx} - \epsilon_i & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} - \epsilon_i & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} - \epsilon_i \end{bmatrix}$$

$$\det() = 0 \Rightarrow \epsilon_i^3 - \bar{I}_1 \epsilon_i^2 + \bar{I}_2 \epsilon_i - \bar{I}_3 = 0 \quad \text{Principal Equation}$$

↳ principal strain $\epsilon_1 \geq \epsilon_2 \geq \epsilon_3$

\bar{I}_1 , \bar{I}_2 and \bar{I}_3 are strain invariants

$$\bar{I}_1 = \text{tr}(\epsilon) = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}$$

$$\bar{I}_2 = \epsilon_{xx}\epsilon_{yy} + \epsilon_{xx}\epsilon_{zz} + \epsilon_{yy}\epsilon_{zz} - \epsilon_{xy}^2 - \epsilon_{xz}^2 - \epsilon_{yz}^2$$

$$\bar{I}_2 = \frac{1}{2} [\text{tr}^2(\epsilon) - \text{tr}(\epsilon^2)], \quad [\epsilon]^2 = [\epsilon][\epsilon]$$

$$\bar{I}_3 = \det([\epsilon])$$

* Then we can obtain principal strain directions $\{s_1\}$, $\{s_2\}$ and $\{s_3\}$

* Then the normalized directions can be obtained $\{s_{n1}\}$, $\{s_{n2}\}$ and $\{s_{n3}\}$
as we learned previously.

Note

$$[Q] = \left[\begin{array}{c} \{s_{n1}\}^T \\ \{s_{n2}\}^T \\ \{s_{n3}\}^T \end{array} \right]$$

row-by-row

↑
transformation
matrix

$$[\bar{\epsilon}] = [Q][\epsilon][Q]^T = \begin{bmatrix} \epsilon_1 & 0 & 0 \\ 0 & \epsilon_2 & 0 \\ 0 & 0 & \epsilon_3 \end{bmatrix}$$

For $[\bar{\epsilon}]$

$$\bar{I}_1 = \epsilon_1 + \epsilon_2 + \epsilon_3$$

$$\bar{I}_2 = \epsilon_1 \epsilon_2 + \epsilon_1 \epsilon_3 + \epsilon_2 \epsilon_3$$

$$\bar{I}_3 = \epsilon_1 \epsilon_2 \epsilon_3$$

* Mean & Deviatoric strains

* For a strain state (or field) called $[E]$

$$[E] = \begin{bmatrix} \epsilon_{xx} & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} \end{bmatrix}$$

* The mean strain (ϵ_M) is defined as:

$$\epsilon_M = \frac{1}{3} \bar{I}_1 = \frac{1}{3} (\epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}) = \frac{1}{3} (\epsilon_1 + \epsilon_2 + \epsilon_3)$$

* Then the mean strain matrix $[E_m]$ is

$$[E_m] = \begin{bmatrix} \epsilon_M & 0 & 0 \\ 0 & \epsilon_M & 0 \\ 0 & 0 & \epsilon_M \end{bmatrix}$$

* Define the deviatoric strain matrix $[E_d]$ as:

$$[E_d] = [E] - [E_m]$$

$$\Rightarrow [E_d] = \begin{bmatrix} \epsilon_{xx} - \epsilon_M & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} - \epsilon_M & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} - \epsilon_M \end{bmatrix}$$

Mean & deviatoric strains are
 Important in failure theories
 "Chapter 4"

• För $[\epsilon_d] = \begin{bmatrix} \epsilon_{xx} - \epsilon_M & \epsilon_{xy} & \epsilon_{xz} \\ \epsilon_{xy} & \epsilon_{yy} - \epsilon_M & \epsilon_{yz} \\ \epsilon_{xz} & \epsilon_{yz} & \epsilon_{zz} - \epsilon_M \end{bmatrix}$ ②

we can find the principal values ($\epsilon_{di}, i=1,2,3$) and principal Directions $\{S_{di}\}$ ($i=1,2,3$), following the same procedure. Thus

$$\epsilon_d^3 - \bar{J}_1 \epsilon_d^2 + \bar{J}_2 \epsilon_d - \bar{J}_3 = 0 \Rightarrow \text{principal values of } [\epsilon_d]$$

$$\epsilon_{d1} \geq \epsilon_{d2} \geq \epsilon_{d3}$$

\bar{J}_1 , \bar{J}_2 and \bar{J}_3 are the deviatoric strain invariants:

$$\bar{J}_1 = \text{tr}(\epsilon_d) = 0$$

$$\bar{J}_2 = \frac{1}{2} [\text{tr}^2(\epsilon_d) - \text{tr}(\epsilon_d^2)] \quad \text{and} \quad [\epsilon_d]^2 = [\epsilon_d][\epsilon_d]$$

$$= -\frac{1}{6} \left[(\epsilon_{xx} - \epsilon_M)^2 + (\epsilon_{xx} - \epsilon_{zz})^2 + (\epsilon_{yy} - \epsilon_{zz})^2 + 6(\epsilon_{xy}^2 + \epsilon_{xz}^2 + \epsilon_{yz}^2) \right]$$

$$= -\frac{1}{6} \left[(\epsilon_1 - \epsilon_2)^2 + (\epsilon_1 - \epsilon_3)^2 + (\epsilon_2 - \epsilon_3)^2 \right]$$

$$= \bar{I}_2 - \frac{1}{3} \bar{I}_1^2$$

$$\bar{J}_3 = \det(\epsilon_d) = (\epsilon_1 - \epsilon_M)(\epsilon_2 - \epsilon_M)(\epsilon_3 - \epsilon_M)$$

$$= \bar{I}_3 - \frac{1}{3} \bar{I}_1 \bar{I}_2 + \frac{2}{27} \bar{I}_1^3$$

Thus, the principal values of $[Ed]$, are

$$\begin{aligned}
 ① \quad \epsilon_{d1} &= \epsilon_1 - \epsilon_M = \frac{(\epsilon_1 - \epsilon_3) + (\epsilon_1 - \epsilon_2)}{3} \\
 ② \quad \epsilon_{d2} &= \epsilon_2 - \epsilon_M = \frac{(\epsilon_2 - \epsilon_3) + (\epsilon_2 - \epsilon_1)}{3} \\
 ③ \quad \epsilon_{d3} &= \epsilon_3 - \epsilon_M = \frac{(\epsilon_3 - \epsilon_1) + (\epsilon_3 - \epsilon_2)}{3}
 \end{aligned} \quad \Rightarrow \quad \epsilon_{di} = \epsilon_i - \epsilon_M$$

From above equations $\Rightarrow \epsilon_{d1} + \epsilon_{d2} + \epsilon_{d3} = 0$

* For principal Directions $\{S_{di}\}$

$$\{S_{di}\} = \{S_i\}, \text{ normalized } \{S_{ni}\} = \{S_{ni}\}$$

= The principal directions of $[Ed]$ are
 the same as those of $[E]$ "

* Octahedral strains

(4)

For an octahedral plane with $\vec{N} = \frac{1}{\sqrt{3}} (\hat{i} + \hat{j} + \hat{k})$

The engineering octahedral shear strain (γ_{oct}), is:

$$\begin{aligned}\gamma_{oct} &= \frac{2}{3} \sqrt{(\epsilon_1 - \epsilon_2)^2 + (\epsilon_1 - \epsilon_3)^2 + (\epsilon_2 - \epsilon_3)^2} \\ &= \frac{2}{3} \sqrt{(\epsilon_{xx} - \epsilon_{yy})^2 + (\epsilon_{xx} - \epsilon_{zz})^2 + (\epsilon_{yy} - \epsilon_{zz})^2 + 6(\epsilon_{xy}^2 + \epsilon_{xz}^2 + \epsilon_{yz}^2)} \\ &= 2 \sqrt{\frac{2}{9} \bar{I}_1^2 - \frac{2}{3} \bar{I}_2}\end{aligned}$$

where, the octahedral shear strain (ϵ_{oct}^s), is

$$\epsilon_{oct}^s = \frac{1}{2} \gamma_{oct}$$

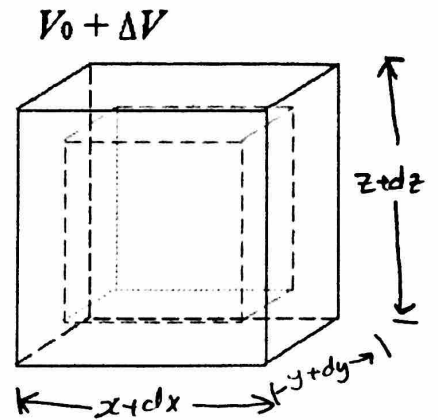
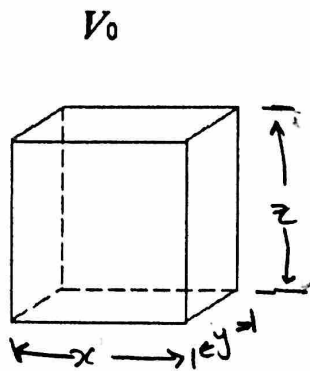
The octahedral normal strain (ϵ_{oct}^N), is

$$\begin{aligned}\epsilon_{oct}^N &= \frac{1}{3} \bar{I}_1 = \frac{1}{3} (\epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}) \\ &= \frac{1}{3} (\epsilon_1 + \epsilon_2 + \epsilon_3)\end{aligned}$$

* Volumetric Strain (ϵ_v) - Dilatation

Dilatation is the relative change of the volume

$$\epsilon_v = \frac{\Delta V}{V_0} = \frac{V_{new} - V_0}{V_0}$$



$V_0 = (x)(y)(z)$ - Divide by $xyz \Rightarrow V_0 = 1$

$V_{new} = (x+dx)(y+dy)(z+dz) \Rightarrow$ divide by $xyz \Rightarrow$

$$= (1 + \epsilon_{xx})(1 + \epsilon_{yy})(1 + \epsilon_{zz})$$

$$\epsilon = \frac{\delta}{L}$$

$$\Rightarrow \epsilon_v = \frac{(1 + \epsilon_{xx})(1 + \epsilon_{yy})(1 + \epsilon_{zz}) - 1}{1}$$

$$= 1 + \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz} + \cancel{\epsilon_{xx}\epsilon_{yy}} + \cancel{\epsilon_{xx}\epsilon_{zz}} + \cancel{\epsilon_{yy}\epsilon_{zz}} + \cancel{\epsilon_{xx}\epsilon_{yy}\epsilon_{zz}} - 1$$

small displacement theory

$$\Rightarrow \epsilon_v = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}$$

$$= \epsilon_1 + \epsilon_2 + \epsilon_3$$

$$= \bar{I}_1$$