

## Chapter 5: Design for vibration suppression

In this chapter, we will only discuss 5.3 vibration Absorbers and this will be the end of the class.

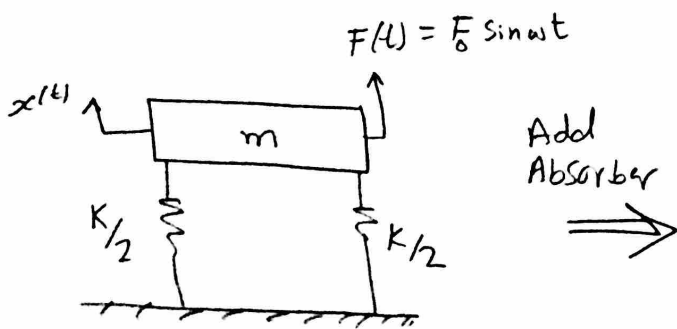
### \* 5.3 Vibration Absorbers :-

\* Vibration absorbers are commonly used to prevent a primary device vibration

\* Vibration absorbers are a second mass-spring system that are attached to the primary device

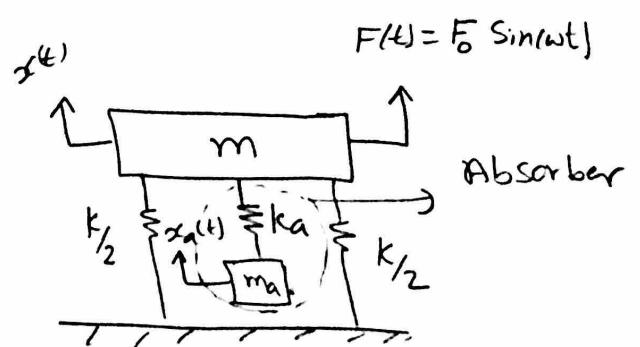
\* The major effect of adding a second spring-mass system is to change from a single-degree-of-freedom system to a two-degree-of-freedom system with two natural frequencies.

\* The values of the mass ( $m_a$ ) and stiffness ( $k_a$ ) of the absorber are chosen such that the motion of the original mass ( $m$ ) is reduced



Primary System (SDoF)

Figure 1



system with absorber (MDoF)

Figure 2

The equation of motion of the 2DOF system of

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Figure 2, is:

$$[m]\{\ddot{x}\} + [k]\{x\} = \{F(t)\} \quad - (1)$$

where

$$[m] = \begin{bmatrix} m & 0 \\ 0 & m_a \end{bmatrix} \quad \left( \begin{array}{l} m: \text{device mass} \\ m_a: \text{absorber mass} \end{array} \right) \quad \circ \quad [k] = \begin{bmatrix} k+k_a & -k_a \\ k_a & k_a \end{bmatrix} \quad \left( \begin{array}{l} k: \text{device stiffness} \\ k_a: \text{Absorber stiffness} \end{array} \right)$$

$$\{\ddot{x}\} = \begin{Bmatrix} \ddot{x}(t) \\ \ddot{x}_a(t) \end{Bmatrix} \quad \left( \begin{array}{l} \ddot{x}(t): \text{Device Acceleration} \\ \ddot{x}_a(t): \text{Absorber acceleration} \end{array} \right) \quad \circ \quad \{x(t)\} = \begin{Bmatrix} x(t) \\ x_a(t) \end{Bmatrix} \quad \left( \begin{array}{l} \text{Device motion} \\ \text{Absorber motion} \end{array} \right)$$

$$\{F(t)\} = \begin{Bmatrix} F_0 \sin \omega t \\ 0 \end{Bmatrix} \quad \rightarrow \text{Applied force on device}$$

To solve this equation

$$\{x(t)\} = \begin{Bmatrix} x(t) \\ x_a(t) \end{Bmatrix} = \begin{Bmatrix} X \sin \omega t \\ X_a \sin \omega t \end{Bmatrix} \quad \rightarrow \text{substitute in eq(1) and re-arrange}$$

$$\begin{bmatrix} k+k_a - m\omega^2 & -k_a \\ -k_a & k_a - m_a\omega^2 \end{bmatrix} \begin{Bmatrix} X \\ X_a \end{Bmatrix} \sin \omega t = \begin{Bmatrix} F_0 \\ 0 \end{Bmatrix} \sin \omega t$$

$$[A] \{x\} = \{F\}$$

$$\Rightarrow \{x\} = [A]^{-1} \{F\}$$

$$\Rightarrow \begin{Bmatrix} X \\ X_a \end{Bmatrix} = \frac{1}{(k+k_a - m\omega^2)(k_a - m_a\omega^2) - k_a^2} \begin{Bmatrix} (k_a - m_a\omega^2) F_0 \\ k_a F_0 \end{Bmatrix}$$

$$X = \frac{(k_a - m\omega^2)F_0}{(k + k_a - m\omega^2)(k_a - m\omega^2) - k_a^2} \quad \text{--- eq(2.a)}$$

$$X_a = \frac{k_a F_0}{(k + k_a - m\omega^2)(k_a - m\omega^2) - k_a^2} \quad \text{eq(2.b)}$$

The goal of adding the absorber is to make the displacement of the primary mass is zero ( $X=0$ ) ← Eq(2.a)

$$\text{So, } (k_a - m\omega^2)F_0 = 0 \Rightarrow \omega^2 = \frac{k_a}{m_a}$$

apply  $k_a = m_a \omega^2$  in equation (2.b), so

$$X_a = -\frac{F_0}{k_a} \Rightarrow \text{so, } x_a(t) = X_a \sin(\omega t) \Rightarrow x_a(t) = \frac{-F_0}{k_a} \sin(\omega t)$$

Now, let's define the ratio between absorber mass to the primary mass as:-

$$\mu = \frac{m_a}{m} \quad \text{--- Eq(3)}$$

and  $\omega_p = \sqrt{\frac{k}{m}}$  Natural Frequency of the primary device without attaching the absorber

$\omega_a = \sqrt{\frac{k_a}{m_a}}$  Natural Frequency of absorber before adding it to the device.

$$\frac{k_a}{k} = \frac{\omega_a^2 m_a}{\omega_p^2 m} = \frac{\omega_a^2}{\omega_p^2} \mu \Rightarrow \frac{k_a}{k} = \mu \beta^2 \quad \text{--- apply in eq(2.a)}$$

$$\beta^2 = \frac{\omega_a^2}{\omega_p^2} \quad \text{or } \beta = \frac{\omega_a}{\omega_p}$$

$$\frac{x_k}{F_0} = \frac{1 - \omega^2/\omega_n^2}{\left[1 + \mu\beta^2 - \left(\frac{\omega}{\omega_p}\right)^2\right] \left[1 - \left(\frac{\omega}{\omega_n}\right)^2 - \mu\beta^2\right]}$$

where  $\omega_n$  = natural frequency of the system with absorber

where

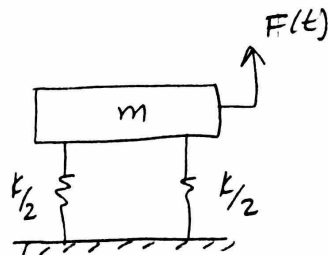
$$\left(\frac{\omega_n}{\omega_n}\right)^2 = \frac{1 + \beta^2(1 + \mu)}{2\beta^2} \pm \frac{1}{2\beta^2} \sqrt{\beta^4(1 + \mu)^2 - 2\beta^2(1 - \mu) + 1}$$

Example :-

If  $m = 73.16 \text{ kg}$

$K = 2600 \text{ N/m}$

$F(t) = 13 \sin(18.85t)$



Design the absorber (Find  $k_a$  and  $m_a$ ) to have  $X_a = 2 \times 10^{-3} \text{ m}$

Soluti

$$X_a = -\frac{F_0}{k_a} \Rightarrow k_a = \left| \frac{F_0}{X_a} \right| = \frac{13}{2 \times 10^{-3}} \Rightarrow k_a = 6500 \text{ N/m}$$

↑ stiffness +ve

also,

$$\omega_a^2 = \frac{k_a}{m_a} \Rightarrow m_a = \frac{k_a}{\omega_a^2} = \frac{6500}{(18.85)^2}$$

$$\Rightarrow m_a = 18.29 \text{ kg}$$