

# Natural Frequencies and modeshapes of Forced and damped MDOF systems

In this class we will consider obtaining natural frequencies and modeshapes of forced and/or damped Multidegree-of-freedom (MDOF) systems. Cases are:

- undamped and forced MDOF
- Damped and unforced MDOF
- Damped and forced MDOF

## ① Undamped and Forced MDOF

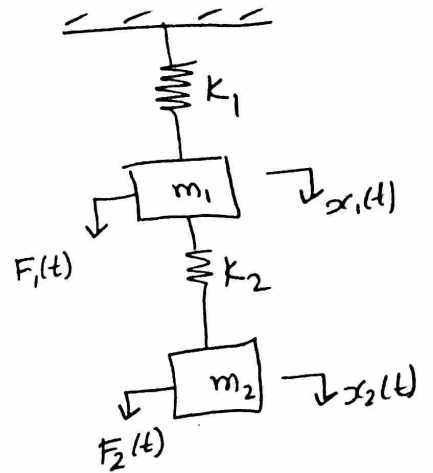
Equation of motion

$$[m] \{\ddot{x}\} + [k] \{x\} = \{F(t)\}$$

where  $[m] = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}$ ,  $[k] = \begin{bmatrix} k_1+k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix}$

$$\{\ddot{x}(t)\} = \begin{Bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \end{Bmatrix}, \quad \{x(t)\} = \begin{Bmatrix} x_1(t) \\ x_2(t) \end{Bmatrix}$$

$$\{F(t)\} = \begin{Bmatrix} F_1(t) \\ F_2(t) \end{Bmatrix} \quad \begin{aligned} F_1(t) &= F_1 \cos \omega t \\ F_2(t) &= F_2 \cos \omega t \end{aligned}$$



## ② Damped and unforced MDOF

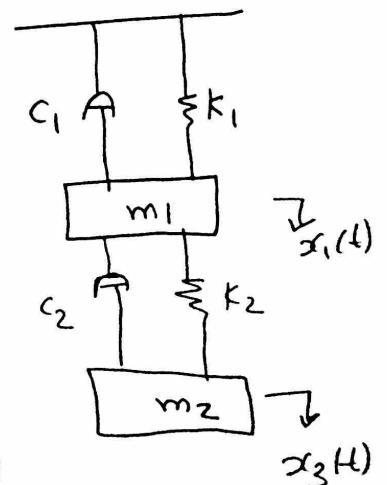
Equation of motion

$$[m] \{\ddot{x}\} + [c] \{\dot{x}\} + [k] \{x\} = \{0\}$$

where  $[m] = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}$  and  $[c] = \begin{bmatrix} c_1+c_2 & -c_2 \\ -c_2 & +c_2 \end{bmatrix}$

$$[k] = \begin{bmatrix} k_1+k_2 & -k_2 \\ -k_2 & +k_2 \end{bmatrix}$$

$[m]$  → mass  
 $[c]$  → Damping  
 $[k]$  → Stiffness



$$\{\ddot{x}(t)\} = \begin{Bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \end{Bmatrix}, \quad \{\dot{x}(t)\} = \begin{Bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{Bmatrix}, \quad \{x(t)\} = \begin{Bmatrix} x_1(t) \\ x_2(t) \end{Bmatrix}$$

# Damped and Forced MDOF

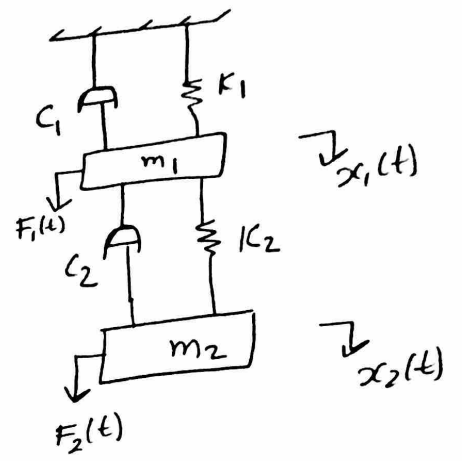
## Equation of motion

$$[m]\{\ddot{x}\} + [c]\{\dot{x}\} + [k]\{x\} = \{F(t)\}$$

where:  $[m] = \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix}$ ,  $[c] = \begin{bmatrix} c_1+c_2 & -c_2 \\ -c_2 & c_2 \end{bmatrix}$

$$[k] = \begin{bmatrix} k_1+k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix}$$

$$\{\ddot{x}(t)\} = \begin{Bmatrix} \ddot{x}_1(t) \\ \ddot{x}_2(t) \end{Bmatrix}, \{\dot{x}(t)\} = \begin{Bmatrix} \dot{x}_1(t) \\ \dot{x}_2(t) \end{Bmatrix}, \{x(t)\} = \begin{Bmatrix} x_1(t) \\ x_2(t) \end{Bmatrix}$$



$$\{F(t)\} = \begin{Bmatrix} F_1(t) \\ F_2(t) \end{Bmatrix}, \begin{matrix} F_1(t) = F_1 \cos \omega t \\ F_2(t) = F_2 \cos \omega t \end{matrix}$$

In All cases above, If we want to find natural Frequencies and modeshapes, we drop (or remove) any damping and forcing terms so the equation of motion would become:-

$$[m]\{\ddot{x}\} + [k]\{x\} = \{0\}$$

then we can find  $\omega_i$  ← natural Frequencies  
 $\{X\}$  ← Modeshapes

as we did previously

\* Note: We will not consider finding  $x_1(t)$  and  $x_2(t)$  for damped/forced MDOF systems.

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3DOF

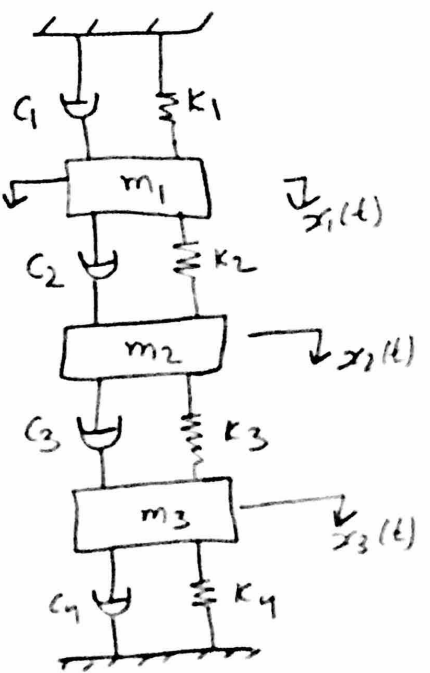
the equation of motion of this system is

$$[m]\{\ddot{x}\} + [c]\{\dot{x}\} + [k]\{x\} = \{F(t)\}$$

where  $[m] = \begin{bmatrix} m_1 & 0 & 0 \\ 0 & m_2 & 0 \\ 0 & 0 & m_3 \end{bmatrix}$ ,  $[c] = \begin{bmatrix} c_1+c_2 & -c_2 & 0 \\ -c_2 & c_2+c_3 & -c_3 \\ 0 & -c_3 & c_3+c_4 \end{bmatrix}$ ,  $F_1(t)$

$$[k] = \begin{bmatrix} k_1+k_2 & -k_2 & 0 \\ -k_2 & k_2+k_3 & -k_3 \\ 0 & -k_3 & k_3+k_4 \end{bmatrix}, \quad \{F(t)\} = \begin{Bmatrix} F_1(t) \\ 0 \\ 0 \end{Bmatrix}, \quad F_1(t) = \cos 3t$$

and  $m_1 = 10 \text{ kg}$ ,  $m_2 = 5 \text{ kg}$ ,  $m_3 = 1 \text{ kg}$   
 $c_1 = 1000 \text{ kg/s}$ ,  $c_2 = 600 \text{ kg/s}$ ,  $c_3 = c_4 = 500 \text{ kg/s}$   
 $k_1 = 500 \text{ N/m}$ ,  $k_2 = 300 \text{ N/m}$ ,  $k_3 = k_4 = 250 \text{ N/m}$



\* Find natural Frequencies and modeshapes of this system.

Solution

\* To Find Natural Frequencies and modeshapes  $\Rightarrow$  Drop  $[c]$  and  $\{F\}$

so,

$$[m]\{\ddot{x}\} + [k]\{x\} = \{0\}$$

As we did previously

$$\{x(t)\} = \{X\} \sin(\omega t + \phi)$$

substitute

$$\Rightarrow ([k] - \omega^2 [m])\{X\} = 0$$

$$\hookrightarrow \det(\ ) = 0 \Rightarrow$$

$$\det \begin{bmatrix} 800 - \omega^2 10 & -300 & 0 \\ -300 & 550 - \omega^2 5 & -250 \\ 0 & -250 & 500 - \omega^2 \end{bmatrix} = 0$$

$$\Rightarrow \omega_1^2 = 39, \quad \omega_2^2 = 121, \quad \omega_3^2 = 530$$

$$\omega_1 = 6.25 \text{ rad/s}, \quad \omega_2 = 11 \text{ rad/s}, \quad \omega_3 = 23 \text{ rad/s}$$

Find mode shapes we apply  $\omega_1^2, \omega_2^2$  and  $\omega_3^2$  in

$$([K] - \omega^2 [m]) \{X\} = 0$$

these are normalized modes shapes

To find

$$\{X^{(1)}\} = \begin{Bmatrix} 0.2242 \\ -0.3062 \\ -0.1662 \end{Bmatrix}$$

$$\{X^{(2)}\} = \begin{Bmatrix} 0.2229 \\ 0.3043 \\ 0.2007 \end{Bmatrix} \rightarrow \{X^{(3)}\} = \begin{Bmatrix} 0.0077 \\ -0.1160 \\ 0.9654 \end{Bmatrix}$$

$$\Rightarrow [U_n] = \begin{bmatrix} 0.2242 & 0.2229 & 0.0077 \\ -0.3062 & 0.3043 & -0.1160 \\ -0.1662 & 0.2007 & 0.9654 \end{bmatrix}$$

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the end of chapter 4